## Moment of inertia and torsional vibrations

(Item No.: P2133100)

## Curricular Relevance



## Difficulty



Difficult

Preparation Time


10 Minutes

Execution Time


20 Minutes

Additional Requirements:

## Keywords:

Rigid body, moment of inertia, axis of rotation, torsional vibration, spring constant, angular restoring moment, moment of inertia of a sphere, moment of inertia of a disc, moment of inertia of a cylinder, moment of inertia of a long bar, moment of inertia of 2 point masses

## Introduction

## Overview

Various bodies perform torsional vibrations about axes through their centres of gravity. The vibration period is measured and the moment of inertia determined from this.


## Equipment

| Position No. | Material | Order No. | Quantity |
| :--- | :--- | :--- | :--- |
| 1 | Rotation axle | $02415-01$ | 1 |
| 2 | Sphere | $02415-02$ | 1 |
| 3 | Disk | $02415-03$ | 1 |
| 4 | Hollow cylinder | $02415-04$ | 1 |
| 5 | Solid cylinder | $02415-05$ | 1 |
| 6 | Rod with movable masses | $02415-06$ | 1 |
| 7 | Spring balance 2,5 N | $03060-02$ | 1 |
| 8 | Light barrier with counter | $11207-30$ | 1 |
| 9 | Power supply 5 V DC/2.4 A with 4 mm plugs | $11076-99$ | 1 |
| 10 | Tripod base PHYWE | $02002-55$ | 1 |
| 11 | Barrel base PHYWE | $02006-55$ | 1 |

## Tasks

The following will be determined:

1. The angular restoring moment of the spiral spring.
2. The moment of inertia

- of a disc, two cylinder, a sphere and a bar,
- of two point masses, as a function of the perpendicular distance to the axis of rotation. The centre of gravity lies in the axis of rotation.


## Set-up and procedure

The experimental set-up is arranged as shown in Fig. 1. To determine the angular restoring moment, the bar is clamped in the torsion spindle and the two masses fixed symmetrically at a defined distance from the axis of rotation. With the spring balance, the bar is rotated by $180^{\circ}$ about the axis in each case and the force is measured. During this measurement, the spring balance is held at right angles to the lever arm. To measure the vibration period of the various bodies, a mask (width $\leq 3 \mathrm{~mm}$ ) is stuck on. The light barrier is pushed over the mask while the body is at rest. Switch the light barrier to 4 -mode. Now the body is deflected through about $180^{\circ}$.

In each case, the time of a half-cycle is measured, several measurements being averaged. For reasons of safety and stability, it is recommended that the spring should not be twistedt beyond $\pm 720^{\circ}$.

## Theory and evaluation

The relationship between the_angular momentum $\vec{L}$ of a rigid body in a stationary coordinate system with its origin at the centre of gravity, and the moment $\vec{T}$ acting on it, is

$$
\begin{equation*}
\vec{T}=\frac{d}{d t} \vec{L} \tag{1}
\end{equation*}
$$

The angular momentum is expressed by the angular velocity $\vec{\omega}$ and the inertia tensor $\hat{I}$ :

$$
\vec{L}=\hat{I} \otimes \vec{\omega}
$$

that is, the reduction of the tensor with the vector. In the present case, $\vec{\omega}$ has the direction of a principal inertia axis (z-axis), so that $L$ has only one component:

$$
L_{Z}=I_{Z} \cdot \omega
$$

where $I_{Z}$ is the z-component of the principal inertia tensor of the body. For this case, equation (1) reads:

$$
T_{Z}=I_{Z} \frac{d \omega}{d t}=I_{Z} \frac{d^{2} \Phi}{d t^{2}}
$$

where $\Phi$ is the angle of rotation.
The moment of a spiral spring, in the Hooke's law range, is:

$$
\begin{equation*}
T_{Z}=-D \cdot \Phi \tag{2}
\end{equation*}
$$

where $D$ is the angular restoring constant.
From the regression line to the measured values of Fig. 2 with the linear statement

$$
Y=A+B \cdot X
$$

the slope

$$
B=0.0265 \mathrm{Nm} / \mathrm{rad}
$$

is obtained.
The angular restoring factor is $D=0.0265 \mathrm{Nm} / \mathrm{rad}$.
The equation of motion reads:

$$
\frac{d^{2} \Phi}{d t^{2}}=\frac{D}{I_{Z}} \Phi=0
$$

The period and frequency of this vibration are respectively

$$
\begin{align*}
& T=2 \pi \sqrt{ } I_{Z} / D  \tag{3}\\
& f=\frac{1}{2} \pi \sqrt{ } D / I_{Z}
\end{align*}
$$

If $\rho(\mathrm{x}, \mathrm{y}, \mathrm{z})$ is the density distribution of the body, the moment of inertia $I_{Z}$ is obtained as

$$
I_{Z}=\iiint\left(x^{2} y^{2}\right) \rho(x, y, z) d x d y d z
$$



Fig. 2: Moment of a spiral spring as a function of angle of rotation.

For a sphere of radius
$r=0.070 \mathrm{~m}$
and of mass $m=0.761 \mathrm{~kg}$
$I_{Z}=\frac{2}{5} m r^{2}=1.49 \cdot 10^{-3} \mathrm{kgm}^{2}$

The measured value is
$I_{Z}=1.48 \cdot 10^{-3} \mathrm{kgm}^{2}$
For a disc of radius
$r=0.108 \mathrm{~m}$
and of mass $m=0.284 \mathrm{~kg}$,
$I_{Z}=\frac{1}{2} m r^{2}=1.66 \cdot 10^{-3} \mathrm{kgm}^{2}$
The measured value is
$I_{Z}=1.68 \cdot 10^{-3} \mathrm{kgm}^{2}$
For a solid cylinder of radius
$r=0.0495 \mathrm{~m}$
and of mass $m=0.367 \mathrm{~kg}$,
$I_{Z}=\frac{1}{2} m r^{2}=0.45 \cdot 10^{-3}{ }^{-3} \mathrm{kgm}^{2}$
The measured value is
$I_{Z}=0.44 \cdot 10^{-3} \mathrm{kgm}^{2}$
For a hollow cylinder with the two radii
$r_{i}=0.070 \mathrm{~m}$
$r_{a}=0.050$
and of mass $m=0.372 \mathrm{~kg}$,
$I_{Z}=\frac{1}{2} m\left(r_{i}^{2}+r_{a}^{2}\right)=0.86 \cdot 10^{-3} \mathrm{kgm}^{2}$
The measured value is
$I_{Z}=0.81 \cdot 10^{-3} \mathrm{kgm}^{2}$

For a thin rod of length
$l=0.6 \mathrm{~m}$
and of mass $m=0.133 \mathrm{~kg}$
$I_{Z}=\frac{1}{12} m l^{2}=3.99 \cdot 10^{-3} \mathrm{kgm}^{2}$
The measured value is
$I_{Z}=3.98 \cdot 10^{-3} \mathrm{kgm}^{2}$


Fig. 3: Moment of inertia of two equal masses, of 0.214 kg each, as a function of the distance between them.
For a point mass of mass $m$, at a distance $a$ from the axis of rotation, one obtains:

$$
\begin{equation*}
I_{Z}=m a^{2} \tag{4}
\end{equation*}
$$

From the regression line to the measured values of Fig. 3, with the statement

$$
\begin{equation*}
Y=A+B X^{2} \tag{4}
\end{equation*}
$$

the slope

$$
B=0.441 \mathrm{~kg}
$$

and the axis intercept

$$
A=0.0043 \mathrm{~kg} / \mathrm{m}^{2}
$$

are obtained.

